

AMENDMENTS TO THE CLAIMS

1. **(Currently Amended)** A hermetic compressor comprising:
a hermetic container for storing oil; and
a compressing element accommodated in the hermetic container for compressing refrigerant gas;
wherein the compressing element has
a compression chamber,
a cylinder forming the compression chamber,
a piston inserted into the cylinder for reciprocating therein, and
a suction muffler having one end communicating with the compression chamber; and
wherein the suction muffler has
a sound deadening space having a first surface,
an inlet pipe having one end opening in the sound deadening space and another end opening to the hermetic container,
an outlet pipe having one end opening in the sound deadening space and another end opening to the compression chamber, said one end opening in the sound deadening space being disposed adjacent the first surface of the sound deadening space,
a gas flow forming part forming a gas flow that enables the gas to flow in a constant direction in the sound deadening space by said one end opening of the outlet pipe disposed adjacent the first surface of the sound deadening space being open so that the gas flowing into the compression chamber from the one end opening of the outlet pipe disposed adjacent the first surface of the sound deadening space flows and circulates in a constant direction along the first surface of the sound deadening space and by opening the one end opening of the inlet pipe at a place which the gas flows into the sound deadening space, and
an oil discharge opening provided in a downstream side of the gas flow in a lower part of the sound deadening space,
wherein the outlet pipe includes a right angle bend in the sound deadening space, the outlet pipe being separate and disconnected from the inlet pipe, and the gas flow forming part is formed by a combination of the outlet pipe and the inlet pipe.

2. **(Previously Presented)** The hermetic compressor of Claim 1, wherein

the gas flow forming part is formed by providing the one end opening in the sound deadening space of the inlet pipe at a thin part of the sound deadening space,

wherein the inlet pipe opens while being extended to any one of an upper end face, a lower end face, a left end face and a right end face of the sound deadening space, thereby constituting the gas flow forming part.

3. **(Previously Presented)** The hermetic compressor of Claim 1, wherein the gas flow forming part is formed by providing the one end opening in the sound deadening space of the outlet pipe at a thin part of the sound deadening space,

wherein the first surface is one of an upper end face, a lower end face, a left end face and a right end face of the sound deadening space, so that the outlet pipe opens while being extended to any one of the upper face, the lower face, the left face and the right face, thereby constituting the gas flow forming part.

4. **(Previously Presented)** The hermetic compressor of Claim 3, wherein a portion of the outlet pipe is disposed adjacent the upper end face of the sound deadening space.

5. **(Previously Presented)** The hermetic compressor of Claim 1, wherein a lower face of the sound deadening space is constituted by a substantially horizontal face, and the oil discharged opening is provided at an end part of the lower face of the sound deadening space.

6. **(Previously Presented)** The hermetic compressor of Claim 1, wherein the suction muffler is formed with an annular gas passage in the sound deadening space.

7. **(Original)** The hermetic compressor of Claim 5, wherein the suction muffler is formed with an annular gas passage in the sound deadening space.

8. **(Previously Presented)** The hermetic compressor of Claim 2,
 wherein the lower end face of the sound deadening space is constituted by a substantially
horizontal face, and the oil discharged opening is provided at an end part of the lower face of the
sound deadening space.

9. **(Previously Presented)** The hermetic compressor of Claim 3,
 wherein the lower end face of the sound deadening space is constituted by a substantially
horizontal face, and the oil discharged opening is provided at an end part of the lower face of the
sound deadening space.

10. **(Previously Presented)** The hermetic compressor of Claim 4,
 wherein the lower end face of the sound deadening space is constituted by a substantially
horizontal face, and the oil discharged opening is provided at an end part of the lower face of the
sound deadening space.

11. **(Previously Presented)** The hermetic compressor of Claim 2,
 wherein the suction muffler is formed with an annular gas passage in the sound
deadening space.

12. **(Previously Presented)** The hermetic compressor of Claim 3,
 wherein the suction muffler is formed with an annular gas passage in the sound
deadening space.

13. **(Previously Presented)** The hermetic compressor of Claim 4,
 wherein the suction muffler is formed with an annular gas passage in the sound
deadening space.

14. **(Previously Presented)** The hermetic compressor of Claim 1, further comprising
a visor, protruding as an eaves, above said oil discharge opening.

15. **(Previously Presented)** The hermetic compressor of Claim 1, wherein

a thin part of the sound deadening space is provided at a lower portion of a central part of the sound deadening space, and

the one end opening in the sound deadening space of the inlet pipe and the one end opening in the sound deadening space of the outlet pipe are provided at the lower portion of the central part of the sound deadening space.

16. **(Currently Amended)** A hermetic compressor comprising:

a hermetic container for storing oil;

a compressing element accommodated in said hermetic container for compressing a refrigerant gas;

said compressing element comprising a cylinder, and a piston disposed in said cylinder for reciprocation, such that a compression chamber is defined by said cylinder and said piston; and

a suction muffler having a sound deadening space therein defined within walls including a top wall, a bottom wall and side walls;

wherein said suction muffler comprises

an inlet pipe, having an internal opening that opens into said sound deadening space and an external opening that opens outside said sound deadening space, for inlet of the refrigerant gas into said sound deadening space,

an outlet pipe, having an internal opening that opens into said sound deadening space and an external opening that opens outside said sound deadening space, for outlet of the refrigerant gas from said sound deadening space, said external opening of said outlet pipe communicating with said compression chamber of said compressing element, and

an oil discharge opening provided at a bottom part of said sound deadening space adjacent one of said side walls such that oil pooled near a junction of said bottom wall and said one of said side walls can discharge through said oil discharge opening,

wherein the gas flowing into the compression chamber from the internal opening of the outlet pipe flows and circulates in a constant direction along one wall of the top wall, the bottom wall and the side walls of the sound deadening space by opening the internal opening of the outlet pipe adjacent to said one wall, and the internal opening of the inlet pipe opens at a place which the gas flows into the sound deadening space so as to constitute a gas flow forming part

that causes a flow of the refrigerant gas along said bottom part of said sound deadening space in a constant direction toward said oil discharge opening to cause the oil in said sound deadening space to pool at said oil discharge opening, and

wherein the outlet pipe includes a right angle bend in the sound deadening space, the outlet pipe being separate and disconnected from the inlet pipe, and the gas flow forming part is formed by a combination of the outlet pipe and the inlet pipe.

17. **(Previously Presented)** The hermetic compressor of claim 16, wherein said at least one of said internal opening of said inlet pipe and said internal opening of said outlet pipe is disposed in a location within said sound deadening space so that said gas flow forming part causes the refrigerant gas to flow along a generally annular path within said sound deadening space.

18 **(Previously Presented)** The hermetic compressor of claim 17, wherein said sound deadening space comprises an upper portion and a lower portion, said lower portion being thinner than said upper portion; and
said lower portion of said sound deadening space has a center portion and side portions on opposing sides of said center portion, said center portion being thinner than said side portions.

19. **(Previously Presented)** The hermetic compressor of claim 16, wherein said sound deadening space comprises an upper portion and a lower portion, said lower portion being thinner than said upper portion; and
said lower portion of said sound deadening space has a center portion and side portions on opposing sides of said center portion, said center portion being thinner than said side portions.

20. **(Previously Presented)** The hermetic compressor of claim 16, further comprising
a visor, protruding as an eaves, above said oil discharge opening.

21-22. **(Cancelled)**